

SUSPENDED SPANS

The prestressed unit is designed and analysed in detail for stress conditions at 2ft. intervals. The Ultimate strength condition is checked and the deflection of the unit under Dead Load plus Superload evaluated.

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100

Prestressing Design of Suspended Span 35' 3" long (22.6 subbeams)

Section Properties at Centre (from computer print out)

- A = 572 m²
- I = 4643 m⁴
- Z_{top} = 1154 m³
- Z_{bot} = 717 m³
- \bar{x}_{top} = 4.02 m
- \bar{x}_{bot} = 6.48 m

Dead load Moment = 976,600 lb-in

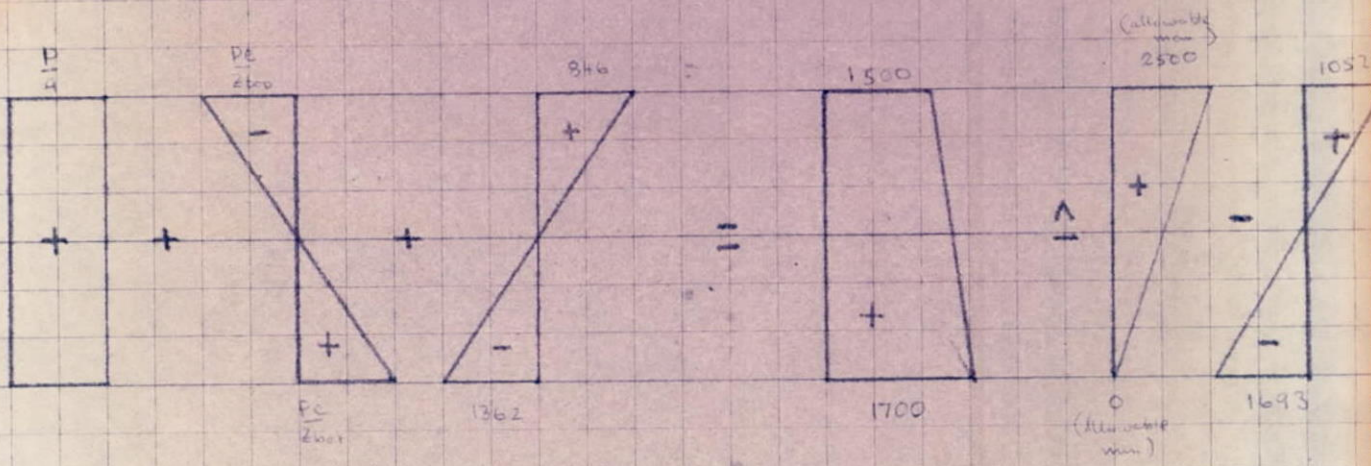
Live load Moment = $\frac{776 \times (22.5)^2 \times 12}{8}$ = 1,213,700 lb-in

Dead load Stresses $\sigma_{top} = \frac{976,600}{1154} = 846$ psi

$\sigma_{bot} = \frac{976,600}{717} = 1362$ psi

Live load Stresses $\sigma_{top} = \frac{1,213,700}{1154} = 1052$ psi

$\sigma_{bot} = 1693$ psi



Equations $\frac{P}{A} + \frac{Pc}{Z_{top}} + 846 = 1500$ - (1)

$\frac{P}{A} + \frac{Pc}{Z_{bot}} - 1362 = 1700$ - (2)

$$\frac{P}{A} - \frac{P_e}{Z_{top}} = 654 \quad - (1)$$

$$\frac{P}{A} + \frac{P_e}{Z_{bot}} = 3062 \quad - (2)$$

$$(1) \times Z_{top} + (2) \times Z_{bot}$$

$$\frac{P(Z_{top})}{A} + \frac{P(Z_{bot})}{A} = 654(Z_{top}) + 3062(Z_{bot})$$

$$P \frac{(Z_{top} + Z_{bot})}{A} = 654(Z_{top}) + 3062(Z_{bot})$$

Sub for Z & simplify

$$P = \frac{(654 \times 1154 + 3062 \times 717) \times 572}{1154 + 717} \quad - (3)$$

$$P = \underline{901,922 \text{ lbf}}$$

Top of E

$$(1) - (2) \text{ + sub for } P \quad e, 901,922 \left[\frac{1}{Z_{top}} + \frac{1}{Z_{bot}} \right] = 3062 - 654 \quad (4)$$

$$e = \frac{2408}{7 \times 16 + 1250} = \underline{1.18}$$

Transfer

$$P_e = \frac{901,922}{0.78} = 1,156,310 \text{ lbf}$$

$$\frac{P}{A} = 2021 \text{ psi}$$

$$\sigma_{bot} = 2021 + 1903 - 1362 = \underline{2562 \text{ psi}}$$

$$\frac{P_e}{Z_{bot}} = 1903$$

However, we have an exact condition to satisfy. Since \bar{x}_{top} at $x = 16.5' \pm 5.4'$ we require an $e \Delta 5.4 - 4.02 = 1.38$ day $1.4'$, to avoid complications at the ends of the beam.

Sub in (2) for e

$$P = \frac{3062}{\left[\frac{1}{A} + \frac{e}{2I_{tot}} \right]} = \frac{3062}{0.001248 + 0.001453} = 1288 \text{ lbs}$$

$$P = \underline{\underline{527,350 \text{ lbf}}}$$

Sub in (1) for P + e

$$\begin{aligned} \sigma_{top} &= 1466 - 1004 + 846 \\ &= 442 \text{ } \cancel{\text{psi}} + 846 \\ &= 1288 \text{ psi} \end{aligned}$$

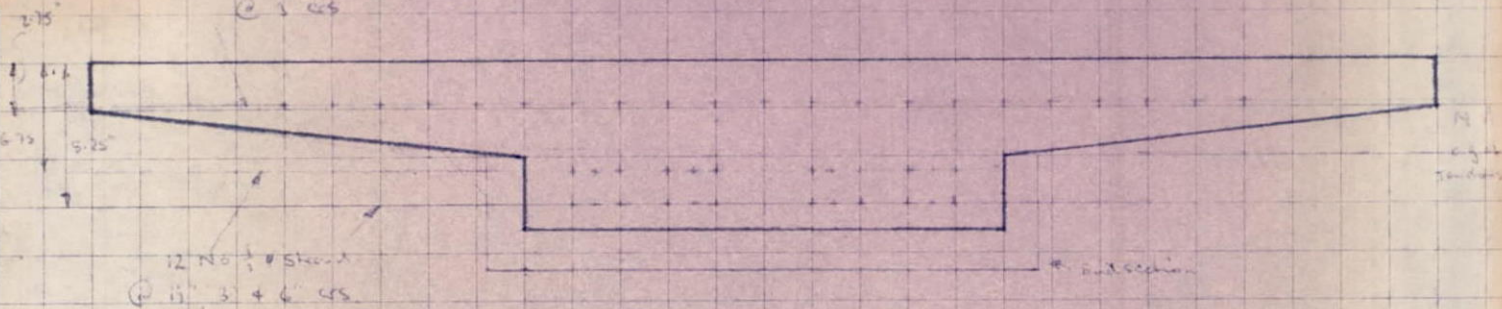
Then final diagram



Pinch $\Delta \frac{527,350}{0.72} = 1,149,000 \text{ lbf}$

Then we require $\frac{1,149,000}{23,900} = 48 \text{ sections } \frac{1}{2} \text{ } \leftarrow$

21 No. 3 channels @ 3 c/s



Distribution:

Two rows of 12 - bottom
One row of 21 - top

$$(1) \quad 21 \times 2.75 + 12 \times x + 12(x+1) = 45 \times 5.42$$

$$(12x + 12x) = 49.90$$

$$x =$$

$$(2) \quad 21 \times 2.75 + 12 \times x + 12(x+24) = 45 \times 5.42$$

$$24x =$$

$$x = 6.97$$

$$(3) \quad 21 \times 2.75 + 24x = 45 \times 5.42 \quad \checkmark$$

$$x = 6.75 (6.) \text{ in } \checkmark$$

Recalculation of Transf. Stresses

$$P = 827,350 \text{ lbf}$$

$$\text{Then } P_t = \frac{827,350}{0.75} = 1,060,705 \text{ lbf}$$

$$\frac{P}{A} = 1854 \text{ psi}$$

$$\text{Then } \sigma_{top} = 1854 - 1297 + 546 = 1412 \text{ psi} \checkmark$$

$$\frac{P_t}{Z_{top}} = 1297$$

$$\sigma_{bot} = 1854 - 2071 - 1862 = 2563 \text{ psi} \checkmark$$

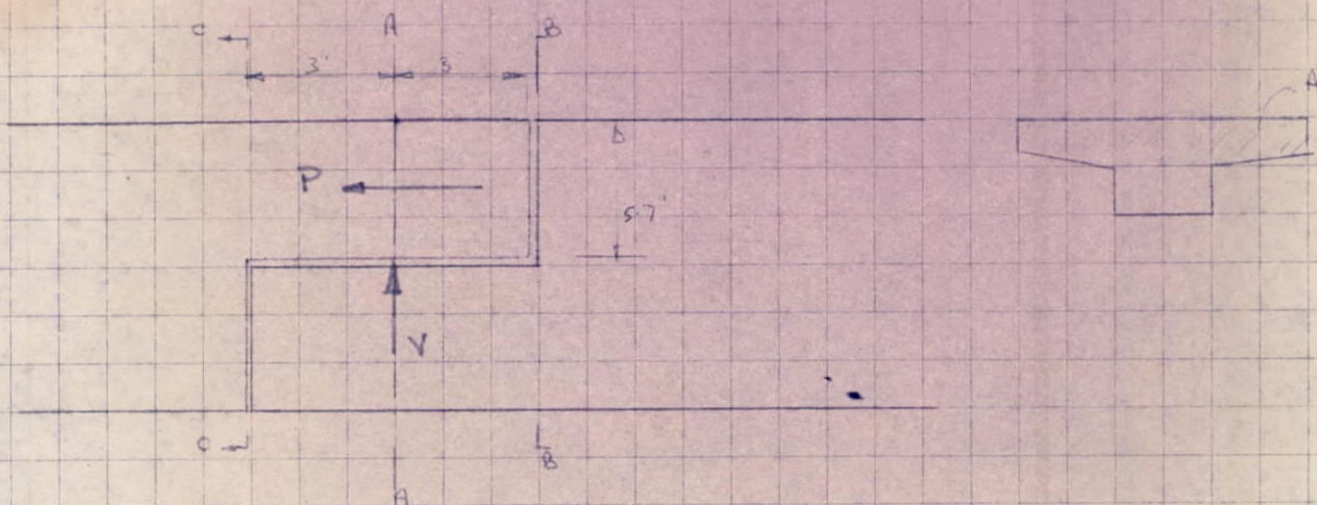
$$\frac{P_t}{Z_{bot}} = 2071$$

Check End Section

$$A = 700 \text{ in}^2$$

$$\frac{P_t}{A} = \frac{1,060,705}{700} = 1515 \text{ psi} \checkmark \left(\frac{P_t}{A} = \sigma_c \text{ ; } e = 0 \right)$$

Shear Stresses in Expansion Joint



$$A = 84 \times 5.7 - 16 \times \frac{5.7}{2} = 443.7 \text{ in}^2$$

Max Shear Force $V = 10,351 + 766 \times 16.25 = 22,800$ lbf

from C & CA research report No 11, transmission length of $\frac{1}{2}$ " strand is about 14" max with a roughly linear build up (graph No 2)

∴ Prestress force at AA = $\frac{3}{162} \times \frac{41^2}{45.55} = 52,350$
 = 52,735 lbf.

allowable principal stress = $\frac{82,735}{2 \times 4437} - \left[\sigma^2 + \left(\frac{52,735}{2 \times 4437} \right)^2 \right]^{\frac{1}{2}} = -160$

~~93,235~~ - $\left[\sigma^2 + 8506 \right]^{\frac{1}{2}} = -258$

$\sigma^2 = 64009 - 8506$
 = 55,503
 $\sigma = 236$ p.s.i.

Allowable shear force = $\frac{236 \times 72 \times \frac{72}{33} \times (5.7)^3}{72 \times 30 \times 15 \times 12} = 58,274$ lbf O.K.
 i.e. > 22,800 lbf

Analyze Section C-C

Prestress = $\frac{6}{14} \times \frac{21}{45} \times 82,350 = 165,470.0$ lbf

R.H.S of section $M_D = 10,351 \times 3 = 31,050$ lb-ft (neg)
 $M_C = 12,450 \times 3 = 37,200$ lb-ft (neg)

Percentage of Tendon Steel at X=0

CL. 314, C.P. 115 requires $\frac{A_{st}}{b \times d} \times 100 > \frac{15}{f_u}$ where: $A_{st} = 6.48 \text{ in}^2$
 $b = 30 \text{ in}$
 $d = 10.5 \text{ in}$
 $f_u = 110 \text{ t/in}^2$

$\frac{A_{st}}{b \times d} = \frac{6.48}{30 \times 10.5} = 2.06\% > \frac{15}{110} = 0.136$

- O.K

Estimate of Prestress Losses - (see C.P.115)

(a) Steel creep.

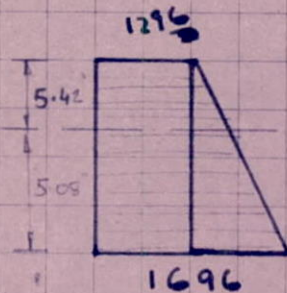
Stress relieved $\frac{1}{2}$ " ϕ strand

10,000 p.s.i.

(b) Elastic deformation

$$\frac{E_s}{E_c} = \frac{28 \times 10^6}{5.25 \times 10^6} = 5.35$$

Now σ_c , Stress in adjacent concrete: 12 at 5.42" from top face.



$$\sigma_c = 1296 + (1696 - 1296) \frac{5.42}{10.5} = 1502 \text{ p.s.i.}$$

$$\therefore \text{loss} = \frac{E_s}{E_c} \times \sigma_c = 5.35 \times 1502 = \underline{8,036 \text{ p.s.i.}}$$

(c) Concrete Shrinkage

$$300 \times 10^{-6} \times 28 \times 10^6 = \underline{8,400 \text{ p.s.i.}}$$

(d) Concrete Creep.

$$0.33 \times 10^{-6} \times 28 \times 10^6 \times 1502 = \underline{13,878 \text{ p.s.i.}}$$

Total Losses 40,314 p.s.i.

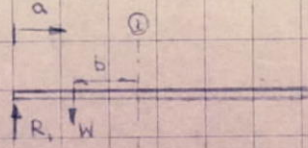
$$\begin{aligned} \text{Required Final Design Force} &= 827,350 \text{ lbf} \\ \text{Stress} &= \frac{827,350}{0.144 \times 45} = 127,677 \text{ p.s.i.} \end{aligned}$$

$$\begin{aligned} \text{Then } \sigma_{\text{initial}} &= 127,677 + 40,314 = 167,991 \text{ p.s.i.} \\ \text{and } P_{\text{initial}} &= 167,991 \times 0.144 \times 45 = 1,088,581 \text{ lbf} \\ \text{and } P/\text{strand} &= \underline{24,200 \text{ lbf}} \end{aligned}$$

$$\% \text{ loss} = \frac{40,314}{167,991} \times 100 = 24\%$$

Stresses in Unit 4

Dead load Moments



$R_1 = 10,350 \text{ lb}$
 $W = \text{Wt. of LHS}$
 $b = \text{c.g. of LHS to C}$

Bending Moment at ① = $R_1 a_1 - Wb$

Wt. at x ft.	Values of x (from Beam centre line) ft															
	0	1	2	3	4	5	6	7	8	9	10	11	12	13	14	15
0.5	3,576	—														
2.0	28,704	14,352														
4.0	57,558	43,416	14,472													
6.0	86,200	73,350	44,150	14,700												
8.0	120,000	105,000	75,000	45,000	15,000											
10.0	154,000	135,750	107,940	77,100	46,260	15,420										
12.0	190,000	174,240	142,580	110,580	79,700	47,570	15,840									
14.0	232,000	215,748	181,336	149,364	116,172	82,950	49,788	16,546								
15.625	166,875	160,200	138,840	117,450	96,110	74,760	53,400	32,040	19,688							
Wb	1,041,900	925,236	705,468	514,524	352,752	220,640	119,028	49,636	10,680							
R.a ₁	2,013,250	1,894,050	1,645,650	1,347,250	1,148,850	900,450	652,050	403,650	155,250							
B.M.	976,350	968,814	940,182	882,726	796,098	679,770	533,022	355,014	144,570							
Δ	0	26,652	57,456	86,618	116,328	146,748	178,008	210,444								
Δ^2	0	-28,652	-28,814	-29,172	-29,750	-30,420	-31,260	-32,436								
Δ^3	0	132	348	528	720	940	1176									
Δ^4	0	-192	-110	-192	-110	-110	-336									
Δ^5																
Δ^6																

NUMERICAL INSTABILITY TO GREAT FOR Δ^5 AND Δ^6

Interpolation for B.M. at even values of x (Newton's forward Difference method)

Formula :-

$$f(x_0 + ph) = f_0 + p \Delta f_0 + \frac{p(p-1)}{2!} \Delta^2 f_0 + \frac{p(p-1)(p-2)}{3!} \Delta^3 f_0 + \dots$$

here, $h = 2$

$x_0 = -15$ (initially)

we require $(x_0 + ph) = -14$ "

whence $p = \frac{1}{2}$

$$\begin{aligned} \text{Then } M_{14} &= \frac{144,570}{\cancel{210,444}} + \frac{1}{2} \cdot 210,444 + \frac{1}{2} \cdot \left(-\frac{1}{2}\right) \cdot (-32,436) + \\ &+ \frac{1}{2} \cdot \left(-\frac{1}{2}\right) \cdot \left(-\frac{3}{2}\right) \cdot 1176 + \dots \end{aligned}$$

$$= \underline{\underline{253,920 \text{ lb.in}}}$$

$$\begin{aligned} M_{12} &= 355,014 + \frac{1}{2} \cdot 176,008 + \frac{1}{8} \cdot 31,260 + \frac{1}{16} \cdot 840 + \dots \\ &= \underline{\underline{447,980 \text{ lb.in}}} \end{aligned}$$

$$\begin{aligned} M_{10} &= 533,022 + \frac{1}{2} \cdot 146,748 + \frac{1}{8} \cdot 30,420 + \frac{1}{16} \cdot 720 + \dots \\ &= \underline{\underline{610,244 \text{ lb.in}}} \end{aligned}$$

$$\begin{aligned} M_8 &= 679,000 + \frac{1}{2} \cdot 116,328 + \frac{1}{8} \cdot 29,700 + \frac{1}{16} \cdot 528 + \dots \\ &= \underline{\underline{740,910 \text{ lb.in}}} \end{aligned}$$

$$\begin{aligned} M_6 &= 796,098 + \frac{1}{2} \cdot 86,628 + \frac{1}{8} \cdot 29,172 + \frac{1}{16} \cdot 348 + \dots \\ &= \underline{\underline{843,081 \text{ lb.in}}} \end{aligned}$$

$$\begin{aligned} M_4 &= 882,726 + \frac{1}{2} \cdot 57,456 + \frac{1}{8} \cdot 28,824 + \frac{1}{16} \cdot 192 + \dots \\ &= \underline{\underline{915,869 \text{ lb.in}}} \end{aligned}$$

$$\begin{aligned} M_2 &= 940,182 + \frac{1}{2} \cdot 28,632 + \frac{1}{8} \cdot 28,632 + 0 + \dots \\ &= \underline{\underline{958,077 \text{ lb.in}}} \end{aligned}$$

Dead Load Stresses

$$\sigma_{top} = \frac{M}{Z_{top}}$$

$$\sigma_{bot} = \frac{M}{Z_{bot}}$$

x ft.	Calculated		Interpolated		σ_{top} p.s.i.	σ_{bot} p.s.i.
	B.M.	lb.in	B.M.	lb.in.		
0	976,350		976,350		+ 846	- 1362
1	968,814					
2			959,077		+ 822	- 1322
3	940,182					
4			915,069		+ 765	- 1229
5	882,726					
6			843,081		+ 577 673	- 1079
7	796,098					
8			740,910		+ 556	- 888
9	679,770					
10			610,244		+ 423	- 672
11	533,022					
12			447,980		+ 289	- 455
13	355,014					
14			253,920		+ 146	- 225
15	144,570					

Prestressing Stresses

Final prestressing force, $P = 827,350$ lbf.

Centroid of tendons to top face = 5.42 ins

\therefore Eccentricity, $e = 5.42 - Y$ (see computer print out)

Prestressing Stresses (Continued)

x ft.	$e = 5.42 - y$ ins	$\frac{Pe}{Z_{top}}$ = σ_b ins	$\frac{Pe}{Z_{bot}}$ = σ_b ins	$\frac{P}{A}$ ins
0	1.396	-1,001	+1611	+1447
2	1.376	-977	+1571	+1442
4	1.324	-915	+1472	+1430
6	1.230	-812	+1302	+1408
8	0.920 1.097	-681	+1087	+1379
10	0.920	-528	+838	+1341
12	0.742	-396	+623	+1305
14	0.429	-203	+315	+1246

Liveload Stresses

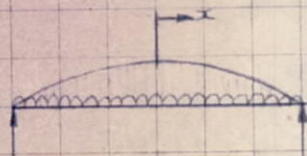
live load is uniform v.d.l of. 766 lb/ft run.
Span 32.5 ft.

Bending Moment equation :-

$$M_x = M_{max} - \frac{wx^2}{2}$$

where $M_{max} = \frac{766 \times 32.5^2 \times 12}{8} = 1,213,700 \text{ lb.in}$

$w = 766 \text{ lb/ft run}$



Liveload Moments and Stresses

$$M_x = 1,213,700 - \frac{766 \times x^2}{2}$$

x ft	Bending Moment ft-lb	σ_{top} p.s.i.	σ_{bot} p.s.i.
0	1,213,700	+ 1052	- 1693
2	1,195,316	+ 1026	- 1650
4	1,140,164	+ 953	- 1532
6	1,049,244	+ 837	- 1342
8	919,556	+ 690	- 1102
10	754,100	+ 523	- 830
12	551,876	+ 356	- 560
14	312,884	+ 178	- 278

STRESSES FOR SUSPENDED SPAN ~ UNIT 4 (Long)

Table E ~ σ_{top} p.s.i.

Col No.	①	②	③	④	⑤	⑥
X ft	Dead Load	$\frac{Pe}{Z_{top}}$	$\frac{P}{A}$	1 + 2 + 3	Live Load	4 + 5
0	+846	-1001	+1447	+1292	+1052	+2344
2	+822	-977	+1442	+1287	+1026	+2313
4	+765	-915	+1430	+1280	+953	+2233
6	+673	-812	+1408	+1269	+837	+2106
8	+556	-681	+1379	+1254	+690	+1944
10	+423	-528	+1341	+1236	+523	+1759
12	+289	-396	+1305	+1198	+356	+1554
14	+146	-203	+1246	+1189	+178	+1367

Table F ~ σ_{bot} p.s.i.

Col No.	①	②	③	④	⑤	⑥
X ft	Dead Load	$\frac{Pe}{Z_{bot}}$	$\frac{P}{A}$	1 + 2 + 3	Live Load	4 + 5
0	-1362	+1611	+1447	+1696	-1643	+3
2	-1322	+1571	+1442	+1691	-1650	+41
4	-1229	+1472	+1430	+1673	-1532	+141
6	-1079	+1302	+1408	+1631	-1342	+289
8	-888	+1087	+1379	+1578	-1102	+476
10	-672	+838	+1341	+1507	-830	+677
12	-455	+623	+1305	+1473	-560	+913
14	-225	+315	+1246	+1336	-278	+1058

Ultimate Strength of Unit 4, Suspended Span 32'-6" long

Load Factors (C.P. 115: 1959)

$$M_u \geq 2(M_D + M_L) \quad \text{where} \quad \begin{aligned} M_u &= \text{Ultimate moment of resistance} \\ M_D &= \text{Dead Load Moment} \\ M_L &= \text{Live Load Moment} \end{aligned}$$

$$\text{Then } M_u > 2(976,600 + 1,213,700) = \underline{\underline{4,380,600 \text{ lb.in}}}$$

For the analysis of this unit, the method as laid down in C.P. 115 is unsuitable, and the method as shown in CCL prestressed Design booklet will be used, treating the unit as a 'T' beam.

Equation:-

$$M_u = K \times A_{stw} \times f_u \times d_1 \left[1 - 0.75 \frac{A_{stw} \times f_u}{b' \times d_1 \times U_w} \right] + 0.7 U_w \times t (b - b') (d_1 - 0.5t)$$

Where :

- $K = 1$ (coefficient of bond efficiency)
- $f_u = 246,000 \text{ lb/in}^2$ (ultimate tendon stress)
- $d_1 = 5.54 \text{ ins}$ (depth from top face to tendon centroid)
- $b' = 30 \text{ ins}$ (breadth of web)
- $b = 72 \text{ ins}$ (width of flange)
- $t = 4 \text{ ins}$ (Average thickness of flange)
- $U_w = 7,500 \text{ p.s.i}$ (28 day concrete strength)
- $A_{stw} = A_{st} - A_{stf}$
- $A_{st} = 6.48 \text{ in}^2$ (Area of steel tendons)
- $A_{stf} = 0.68 U_w (b - b') \frac{t}{f_u}$

$$A_{stf} = 0.68 \times 7,500 (72 - 30) \frac{4}{246,000} = 3.48 \text{ ins}$$

$$\text{Then } A_{stw} = 6.48 - 3.48 = 3 \text{ ins}$$

Then :-

$$\begin{aligned} M_u &= 3 \times 246,000 \times 5.54 \left[1 - \frac{0.75 \times 3 \times 246,000}{30 \times 5.54 \times 7,500} \right] + 0.7 \times 7,500 \times 4 (42) (5.54 - 2) \\ &= 2,289,600 + 9,122,300 \text{ lb.in} \end{aligned}$$

$$\underline{\underline{M_u = 5,411,900 \text{ lb.in}}} \quad - \text{ O.K}$$

Design of Suspended span 31'-4" long (30'-6" L.S. bearings)

Section properties at centre (as for ~~other~~ beam)

$$\begin{aligned} A &= 572 \text{ in}^2 \\ I &= 4643 \text{ in}^4 \\ Z_{top} &= 154 \text{ in}^3 \\ Z_{bot} &= 217 \text{ in}^3 \\ \bar{x}_{top} &= 4.01 \text{ in} \\ \bar{x}_{bot} &= 6.46 \text{ in} \end{aligned}$$

$$\text{Dead Load Moment} = 858,135 \text{ lb.in}$$

$$\text{Live Load Moment} = 766 \times \frac{30.5^2}{8} \times 12 = 1,068,900 \text{ lb.in}$$

$$\begin{aligned} \text{Dead Load Stresses} \quad \sigma_{top} &= 926 \text{ p.s.i.} \\ \sigma_{bot} &= 1197 \text{ p.s.i.} \end{aligned}$$

$$\begin{aligned} \text{Live Load Stresses} \quad \sigma_{top} &= 926 \text{ p.s.i.} \\ \sigma_{bot} &= 1491 \text{ p.s.i.} \end{aligned}$$

We shall try to keep same wire formation, then let $e = 140 \text{ in}$

As before, sub in (2)

$$\frac{P}{A} + \frac{Pe}{Z_{bot}} - 1197 = 1500$$

$$P = \frac{2,697}{\frac{1}{572} + \frac{140}{217}} = \frac{2,697}{0.003191} = 728,730 \text{ lb}$$

We can either (a) use 45 No. $\frac{1}{2}$ " strand at $\frac{728,730}{45 \times 0.72} = 22,490 \text{ lb}$

or (b) use $\frac{728,730}{25,900 \times 0.76} = 40 \text{ No strand}$

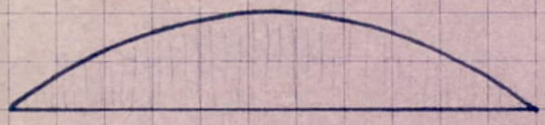
[We have shown in the analysis of Unit 4 that the central section is critical. Therefore No further analysis will be made of unit 2]

(a) After reestimation of losses, $P/\text{strand} = \frac{728,730}{45 \times 0.76} = 21,400 \text{ lb (avg)}$

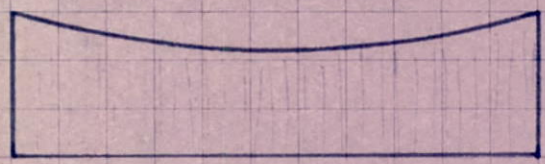
Deflection of Unit 4 under Dead Load and Prestress and Superload

Using Mohr's Theorem

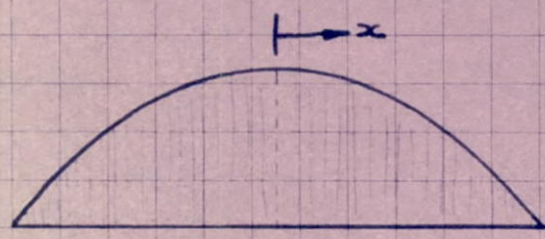
M diag.



I diag.



$\frac{M}{I}$ diagram.



We require moment of $\frac{1}{2} \frac{M}{EI}$ diagram about one end. The diagram will be considered as made up of strips, generally 2ft wide, centered about even values of x.

P = 827,350

x ft	Dead Load B.M. MD lb-in	e ins	P e lb-in	$M_D - P e$ lb-in	Superload Moment M _s lb-in	$M_D - P e + M_s$ lb-in
0	976,350	1.396	1,154,940	-178,630	263,021	84,391
2	958,077	1.376	1,135,433	-180,356	259,037	79,681
4	915,069	1.324	1,095,411	-190,342	247,085	66,743
6	843,081	1.230	1,017,640	-174,559	227,165	52,606
8	740,910	1.097	907,602	-166,692	199,277	32,585
10	610,244	0.920	761,162	-150,918	163,421	12,503
12	447,980	0.742	613,593	-166,613	119,597	-4,7016
14	253,920	0.429	354,933	-101,013	67,805	-33,208
16	2,400	0.003	2,462	-82	-	-

x ft	$M_D - P_e + M_S$ = M lb/ft	I ins ⁴	$\frac{M}{I}$ lb/ins ²	Strip width ins	Area of Strip lb/ins ²	Arm in	Moment about End lb/in
0	84,391	4643	18.18	12"	218.16	139	41,232
2	75,681	4714	16.69	24	400.56	171	69,495
4	66,743	4901	13.62	24	326.88	147	49,051
6	52,606	5250	10.02	24	240.48	123	29,579
8	32,585	5765	5.65	24	135.60	99	13,424
10	12,503	6485	1.93	24	46.32	75	3,474
12	-47,016	7252	-6.48	24	-155.52	51	-7,931
14	-33,208	8705	-3.81	24	-91.44	27	-2,469
16	-	10,362	0	9	0	3	0

$$\Sigma M = 193,855 \text{ lb/in}$$

$$\text{Deflection, } \delta = \frac{\Sigma M}{E_c} = \frac{193,855}{5.75 \times 10^6}$$

(of centre relative to end)

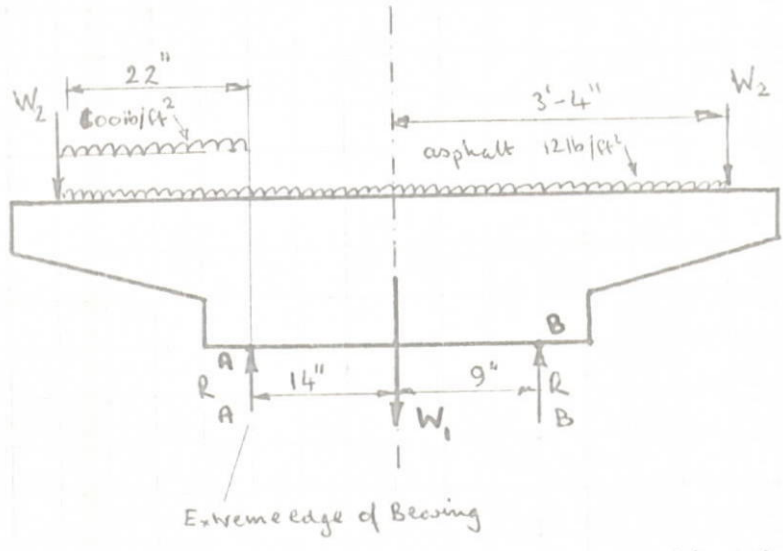
$$\delta = \underline{\underline{+ 0.034''}} \text{ (downwards)}$$

This deflection of $\frac{1}{32}''$ is small enough not to have to consider setting up of formwork.

By examination of the $\frac{M}{I}$ diagrams, it can be seen that unit 2 will have a similar deflection, and no further action need be taken.

Lateral Stability of Suspended Span

32'-6" long.



$$W_1 = \frac{W_t \text{ of beam}}{2} = 10,350 \text{ lbf.}$$

$$\left(\frac{70+14}{2} \right) \times 16 = W_2 = \text{copying + railing} = 2,690$$

$$\text{Asphalt} = 12 \times 6 \times 16 = 1,150$$

$$\Sigma \text{ Load acting on Centreline} = \underline{\underline{14,190 \text{ lbf.}}}$$

T.M.A. A.

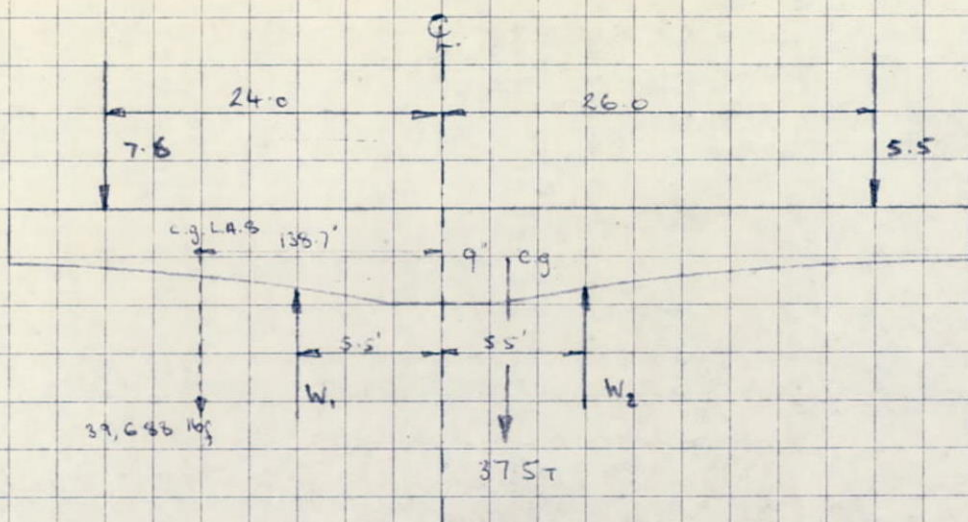
$$R_B \times 23 + 100 \times 2.16 \times 16.25 \times \frac{12}{2} = 14,190 \times 14$$

$$R_B = \underline{\underline{+ 6,810 \text{ lbf.}}}$$

i.e. R_B is +ve. so the beam is stable.

$$\left(\text{Torsional Moment, T} = 6,810 \times \frac{23}{9} = \underline{\underline{154,000 \text{ lb.in}}} \right)$$

Unit 3

T.M.A. W_2

$$(7.6 \times 138.7) - (37.5 \times 6.25) = 5.5 \times 31.5 + W_2 \times 11.0$$

$$W_2 = 24.28 T$$

Vert. Eq'bm.

$$W_1 = 7.6 + 5.5 + 37.5 - 24.28 = 26.32 T$$

B.M. at \bar{x} (consider L.H.S.)

$$M = (7.6 \times 24.0 - 26.32 \times 5.5) \times 2240 \times 12 + 39,688 \times 138.7$$

$$= 6,516,488 \text{ lb.in.}$$

$$\text{check } \sigma_{\text{bot}} = \frac{6,516,488}{14,033.92} - 510 + 431 = +95.3 \text{ psi.}$$

By inspection σ_{top} of Unit 3, and σ_{bot} and σ_{top} of Unit 5 are safe.

Amman